

Improving pumping plant efficiency does not always save energy

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California's energy crisis in 2001 resulted in a state-funded program for testing irrigation pumps and improving pumping plant efficiency, with the goal of reducing energy use in California agriculture. Yet in reality, improving pumping plant efficiency may not actually translate into savings. To reduce electrical energy use, the kilowatt-hours must decrease because of fewer kilowatts or less operating time, or both. In order to evaluate the efficiency of various energy-improving adjustments, we studied several operations at pumping plants in the San Joaquin Valley. These included adjusting impellers, repairing worn pumps, replacing mismatched pumps and using more energy-efficient motors. We found that adjusting or repairing worn pumps may actually increase energy use, unless the operating time of the pumping plant is reduced. Multiple pump tests of a pumping plant are recommended, to help evaluate possible reasons for low efficiency. Pumping plant operators should also obtain the manufacturer's performance curves to use in the evaluation process.

In the late 1980s, the question "Does improving pumping plant efficiency save energy?" was posed to a group of growers and utility company representatives in Salinas. Most of the growers said no, while most of the utility employees said yes. Before this meeting, the utility company had spent a great deal of money to improve the efficiency of irrigation pumping plants in the Salinas area. The goal was to improve the ratio of pump output horsepower to input



Many California farms use pumping plants to facilitate irrigation. In order to save energy, the state has initiated a program to test and improve pump efficiency.

horsepower; in other words, to make the plants more efficient by pumping a greater volume or pressure of water while using the same or less energy.

Pumps are commonly used by California growers to pump water for irrigation. Deep-well turbine pumps are installed in a well for pumping groundwater. They consist of the pump housing or bowl, which contains an impeller (fig. 1). The impeller contains specially shaped vanes to channel the water. The impeller is attached to a shaft, which extends up to the ground surface and is connected to an electric motor or engine. Rotating the shaft, which occurs when the motor or engine is started, causes water to flow into the impeller. In addition, centrifugal or booster pumps generally are used to provide pressure (see photo, page 126). They are located at the ground surface and also consist of a housing, impeller and a short shaft attached to a motor or engine.

Pump efficiency is a measure of the amount of power produced by the pump per unit of input power. The power (rate of energy use) produced by a pump, called the water horsepower, depends on the pump capacity or flow rate, pressure developed by the pump, and the amount of pumping lift in the well (pumping lift is the elevation difference between the dis-

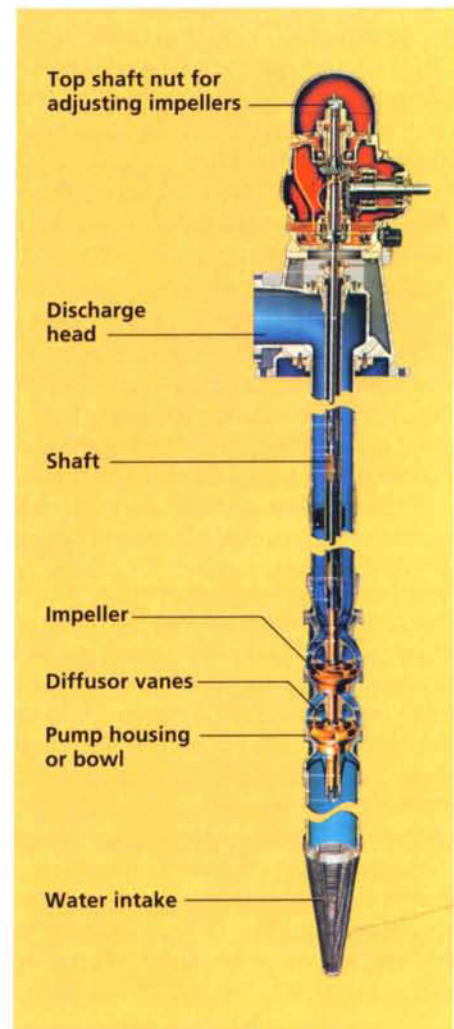


Fig. 1. Parts of a deep-well turbine.

TABLE 1. Effect of impeller adjustment on pumping plant performance

		Capacity (gpm)	Total head (feet)	Overall efficiency (%)	Input horsepower
Pump 1	Before	605	148	54	42
	After	910	152	71	49
Pump 2	Before	708	181	59	55
	After	789	206	63	65
Pump 3	Before	432	302	54	61
	After	539	323	65	67
Pump 4	Before	616	488	57	133
	After	796	489	68	144

TABLE 2. Effect of pump repair on pumping plant performance

	Before	After
Pumping lift (feet)	95	118
Capacity (gpm)	1,552	2,008
Input horsepower	83	89
Overall efficiency (%)	45	67

Calculating pump efficiency

The efficiency of a pumping plant is calculated as follows:

$$E_o = \frac{Q \times H}{3,960 \times IHP}$$

where E_o is the overall pumping efficiency, Q is the pump flow rate or capacity (gallons per minute [gpm]), H is total head or lift (feet), IHP is the input horsepower and 3,960 is a conversion factor, which converts the product of $Q \times H$ into horsepower. The total head is the sum of the pumping lift (elevation difference between the pump discharge pipe and the pumping water level in the well) and the discharge pressure head (discharge pressure in pounds per square inch [psi] multiplied by 2.31). The discharge pressure must be converted to feet of head to make it compatible with pumping lift, which is also in feet. (Note: a column of water 2.31 feet high creates 1 psi of pressure at its base.)

The input horsepower depends on the energy source. For electric motors, IHP can be calculated using

$$IHP = \frac{48.1 \times Kh}{t}$$

where Kh is the meter constant (stamped on the power meter faceplate) and t is the time in seconds for 10 revolutions of the meter disc. Newer power meters display the kilowatt demand of the electric motor. Kilowatts are multiplied by 1.34 to obtain horsepower.

The input horsepower of a diesel engine can be calculated by

$$IHP = 55 \times q$$

where q is the fuel consumption of the engine in gallons per hour. The fuel consumption can be measured by disconnecting the fuel line from the fuel tank, placing it into a container filled with a known volume of fuel and measuring the time it takes to fill the container with fuel. The discharge end of any bypass fuel line should also be inserted into the container. The input horsepower of a diesel engine may be three to four times that of an electric motor because of differences in engine efficiency. However, both are rated based on the brake or shaft horsepower: An electric motor rated at 100 horsepower produces the same power as a 100-horsepower diesel engine.

An inefficient engine may cause low overall efficiency even when the pump itself is efficient. Separating engine efficiency from pump efficiency requires specialized equipment. The efficiency of an electric motor tends to be relatively constant as long as the motor will run — unless the motor becomes severely underloaded, making separation easier.

— B.R.H.

charge of the pump and the pumping water level in the well). The higher the efficiency, the more power that is produced per unit of input power.

Electricity users pay for energy based on the number of kilowatt-hours (kwh) consumed (plus fixed charges). The kilowatt is the power demand of the electric motor; 1 kilowatt equals 1.34 horsepower. The hours are the operating time of the motor. To reduce electrical energy use, the kilowatt-hours must decrease because of fewer kilowatts or less operating time, or both. Regardless of the claims about a proposed energy-saving measure, if the number of kilowatts or the operating time is not reduced, no energy savings will occur.

The 2001 energy crisis resulted in a state-funded program for testing pumps and improving pumping plant efficiency. The goal, of course, was to reduce energy use in California. Yet the grower response at the Salinas meeting suggests that improving efficiency may not actually translate into cost savings. This article discusses the reasons behind the growers' response.

Improving efficiency

Options for improving pumping plant efficiency include adjusting impellers, repairing or replacing worn pumps, replacing mismatched pumps, and converting to energy-efficient electric motors. The effect of these options on energy use was evaluated using data collected over the past 20 years from numerous pumping plant tests conducted by the author, utility companies and

TABLE 3. Pump test data of a mismatched pump

Measurement	Value
Pumping lift (feet)	113
Discharge pressure (psi)	50
Total head (feet)	228
Capacity (gpm)	940
Input horsepower	112
Overall efficiency (%)	48

companies that install and maintain pumps.

Adjusting impellers. Maintaining the appropriate clearance between the bottom of the vanes of a semi-open impeller and the bowl housing is critical for efficient pump performance. Wear caused by sand in the well water can increase the clearance between the impellers and housing, and reduce pumping plant efficiency. Efficiency can be partially restored by adjusting the impellers. This involves slightly lowering the pump shaft and, in turn, the impellers, by rotating the nut at the top of the shaft (fig. 1). This adjustment will not work for enclosed impellers (fig. 2).

When we adjusted the impellers of four pumps, both pump capacity and overall efficiency increased considerably (table 1). Total head increased slightly, because pumping lift only contributed to total head (see box, page 124). However, for all four pumps, impeller adjustments increased input horsepower. Therefore, if the pumps are operated for the same amount of time after the adjustment (a common practice), energy use will increase. Energy use will decrease only if the operating time is decreased by pumping the same volume of water after the adjustment as before. For these data (table 1), the increase in energy costs ranged from 8.3% to 18.2% for the same operating time before and after adjustment. For tests 1, 3 and 4, the decrease in energy costs ranged from 12% to 22.4% to pump the same volume of water. For test 2, energy costs increased by 6%. In this case, the in-

crease in pump capacity was insufficient to offset the increase in input horsepower because part of the increased pump output also contributed to increased total head.

Repairing worn pumps. Repairing a worn pump can increase capacity, total head and overall efficiency, as shown by the pump test data in table 2. However, for this data the input horsepower increased from 83 to 89. This behavior may be typical of many repaired pumps. A summary of 63 data sets of pump performance before and after repair shows increases of 39%, 0.5% and 33% in pump capacity, total head and overall efficiency, respectively (Hanson 1988). The small increase in total head occurred because pumping lift was the main contributor. However, the repair increased the input horsepower for 58% of the pumping plants, with an average increase of 17%. For these pumping plants, using the same operating time before and after the pump repair will increase energy use by 17%. However, pumping the same volume will decrease the average energy use by 22%. For many pumping plants, reducing the operating time may be necessary to realize any energy savings from pump repairs.

Operators of irrigation pumping plants commonly run repaired or adjusted pumps for the same amount of time after repairs or adjustments as they did before. Operating times are often based on the management and

operating requirements of the particular irrigation system; as a result, it may not be possible to reduce operating time in many cases. The practice of not reducing operating times is what led to the growers' response in Salinas; it is possible that most irrigators were unaware of the need to reduce operating times and as a result were using more energy after their pumps were repaired.

Replacing mismatched pumps. A performance characteristic of deep-well turbine and centrifugal (booster) pumps is that as pump capacity increases, pump efficiency increases to a maximum and then decreases. New pumps should be selected to provide the desired flow rate and total head near the point of maximum efficiency, which minimizes the horsepower demand of the pump. Initially efficient pumps can become inefficient because of changes in operating conditions, such as different groundwater levels or alterations in discharge to pressurized irrigation systems, even though the pump is operating properly (no wear).

A pump operating properly but not near the point of maximum efficiency is said to be mismatched to the operating conditions. Used pumps are also candidates for being mismatched. To restore the pumping plant's efficiency, the mismatched pump must be replaced with one providing the desired total head and capacity near maximum efficiency. This change reduces

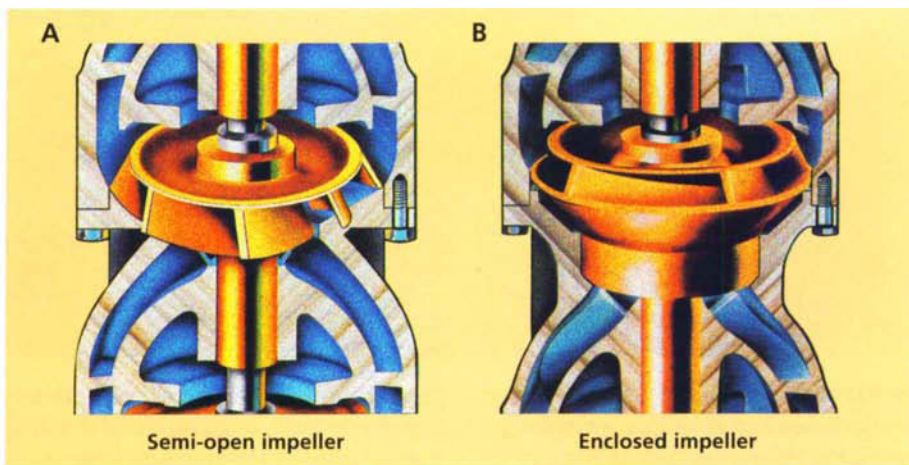
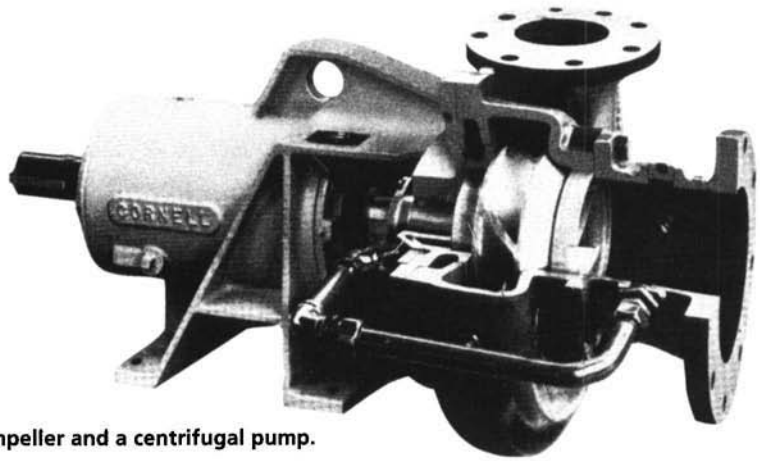


Fig. 2. To improve pump performance, semi-open impellers (A) can be adjusted, but enclosed impellers (B) cannot.



Worn impeller and a centrifugal pump.

the kilowatt demand of the pump and results in energy savings even if the operating time is unchanged.

A review of pump test data from a pump (table 3) revealed an overall efficiency of 48%. However, an efficiency of 57% was found by testing the pump under several different conditions. This relatively high efficiency suggested a mismatched pump. An analysis showed that replacing the inefficient pump with one producing the same output at an efficiency of 60% would reduce the input horsepower from 112 to 90, for a 19.6% reduction in energy use.

Using energy-efficient electric motors. Energy-efficient electric motors need less input horsepower than standard motors (table 4). Buying an energy-efficient motor for a new irrigation pumping plant is more economical than retrofitting an existing pumping plant. For example, an energy-efficient, 100-horsepower motor can cost \$6,000, compared with \$5,000 for a standard motor. The input horsepower of the energy-efficient motor will be 104 compared with 109 for the standard motor. At a typical cost of \$0.1 per kilowatt-hour operating the pump for 2,000 hours per year will save \$746, with a simple payback period of 1.3 years. The payback period for retrofitting, on the other hand, is 8 years.

Evaluating plant performance

Pumping plants should be evaluated every several years to determine the status of the pump and possible reasons for poor efficiency. Evaluating a pumping plant requires a pump test, during which capacity (flow rate), lift, discharge pressure and input horsepower are measured. The overall

pumping plant efficiency is calculated from these data, and can then be compared with the standards for correcting electric pumping plants (table 5)(Hanson 2000).

Pump wear or a mismatched pump can cause poor efficiency. Likewise, repairing a mismatched pump may not improve efficiency. So how can one determine if a pump is mismatched or worn?

One approach is to conduct multiple pump tests, each under different operating conditions. These conditions can be imposed on a pump simply by partially closing a valve in the discharge pipe or changing the number of sprinkler pipelines or drip lines

that are irrigated. An efficiency of about 60% or more for one of the tests indicates a mismatched pump under the normal operating conditions.

A second approach is to compare pump test data under normal operating conditions with the manufacturer's performance data. Manufacturers provide information for each pump on the relationships between capacity and total head, efficiency and brake horsepower. By comparing the total head of the pump test with the manufacturer's total head at the measured capacity, one can evaluate if a low efficiency is due to wear or mismatched operating conditions.

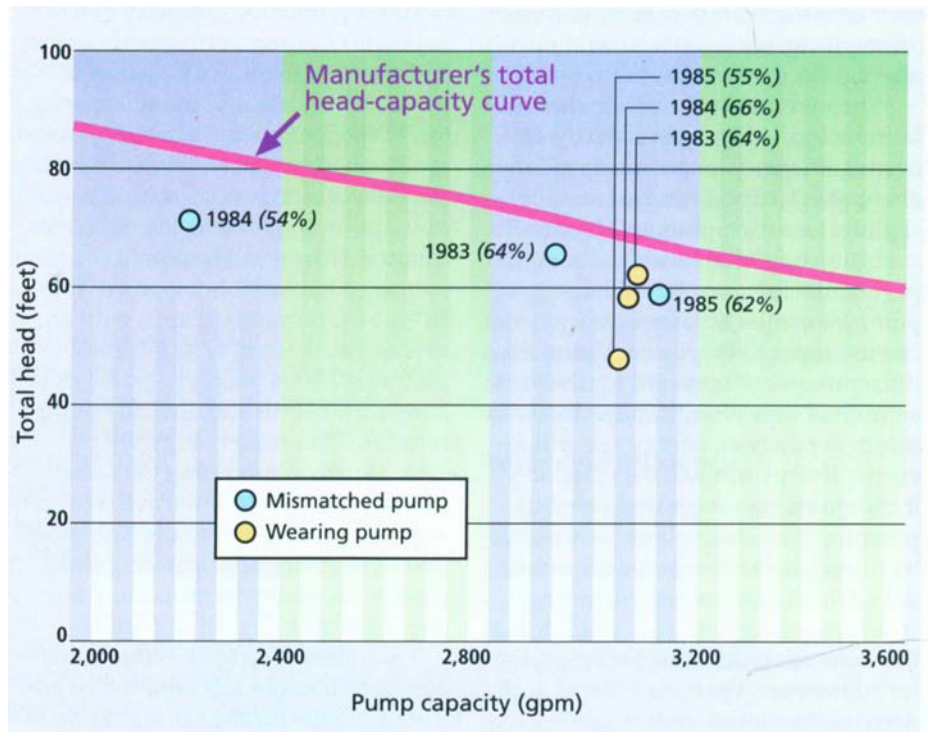


Fig. 3. Comparison of pump test data with manufacturer's total head-capacity curve of a mismatched pump and a wearing pump. Numbers in parentheses are pumping plant efficiency.

TABLE 4. Efficiencies of standard and energy-efficient electric motors

Horsepower	Standard	Energy efficient
%.....	
10	86.5	91.7
20	86.5	93.0
50	90.2	94.5
75	90.2	95.0
100	91.7	95.8
125	91.7	96.2

TABLE 5. Efficiency standards for pumping plants with electric motors

Efficiency range	Suggested corrective action
Eo* greater than 60%	No action
55% to 59%	Consider adjusting impeller
50% to 54%	Consider adjusting impeller; consider repairing or replacing pump
Less than 50%	Consider repairing or replacing pump

* Overall pumping plant efficiency.

For example, a pump test conducted in 1984 showed an efficiency of 54% (fig. 3, mismatched pump). However, the deviation of the test point from the manufacturer's curve was about the same as that of the 1983 test point. The following year (1985), pump efficiency rebounded. This behavior suggests that the 1984 efficiency was due to a mismatched condition. The 1985 point, which shows efficiency similar to the 1983 test, verifies this. However, a second data set showed that the deviation between pump test data and the manufacturer's performance curve increased with time, indicating possible increasing pump wear (fig. 3, wearing pump).

Some caution is necessary in taking this approach. Poor-quality pump test data may prevent an accurate evalua-

tion of the pumping plant. Cascading water in a well may prevent good measurements of pumping lift. Poor test conditions may also prevent good flow-rate measurements. The test section should have eight to 10 pipe diameters of straight pipe immediately upstream of a flow meter and two pipe diameters downstream to prevent errors due to excessive turbulence in the water. For a 10-inch-diameter pipe, a straight section 80 to 100 inches long is recommended. However, research has shown that propeller flow meters are less susceptible to large errors from turbulence caused by bends or elbows and checks valves upstream from the meter (Hanson and Schwankl 1998).

Energy savings

This series of studies shows that simply improving pumping plant effi-

ciency does not guarantee energy savings. In fact, adjusting or repairing worn pumps may increase energy use unless the operating time of the pumping plant is reduced. Sometimes, adjusting the operating time still will not save energy if part of the increase in pump output contributes to a significant total head increase in addition to a capacity increase. With a higher flow rate, pumping plant operators can reduce operating time, with either less irrigation time per set or greater acreage irrigated per set. Opportunities for reducing the operating time will depend on site-specific conditions, such as the irrigation method and its design and management characteristics. If reducing the operating time is not possible, the improved efficiency may result in more crop yield and revenue due to more water applied to a field. Both growers and others apparently did not recognize this fact at the Salinas meeting.

Multiple pump tests (at least three) of a pumping plant are recommended, to help evaluate possible reasons for low efficiency. Pumping plant operators should also obtain the manufacturer's performance curves to use in the evaluation process.

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Adjustments to impellers, made by turning the nut at the top of the pump shaft, can improve pump efficiency. But that may not necessarily translate into energy savings, unless operating time is reduced.